

Problem 1.

[10 points]

Sketch the Bode plot of the following transfer function:

$$G(s) = \frac{2}{s^2 + 22s + 40}$$

Label the x - and y -axis of both the magnitude and phase plots carefully.

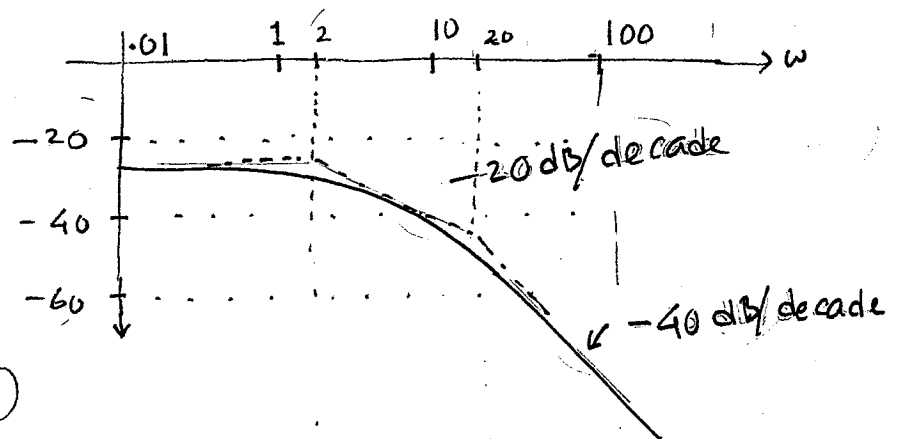
$$G(s) = \frac{2}{(s+2)(s+20)} \Rightarrow G(j\omega) = \frac{2}{(j\omega+2)(j\omega+20)} = \frac{\frac{1}{20}}{\left(\frac{j\omega}{2} + 1\right)\left(j\frac{\omega}{20} + 1\right)}$$

$$20 \log_{10} |G(j\omega)| = 20 \log_{10} \frac{1}{20} - 20 \log_{10} \left| \left(\frac{j\omega}{2} + 1\right) \right| - 20 \log_{10} \left| j\frac{\omega}{20} + 1 \right|$$

$$= -26.02 - 20 \log_{10} \left| \frac{j\omega}{2} + 1 \right| - 20 \log_{10} \left| \frac{j\omega}{20} + 1 \right|$$

\uparrow breakpoint at $\omega=2$ \uparrow breakpoint at $\omega=20$

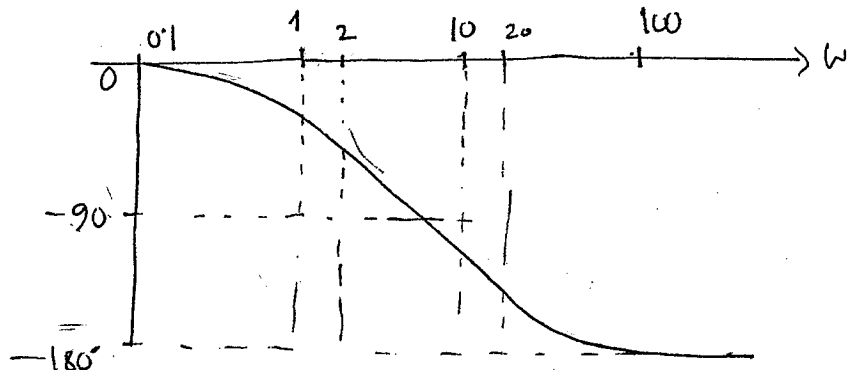
$|G(j\omega)|$
(dB)



$$\angle G(j\omega) = 0 - \angle \left(\frac{j\omega}{2} + 1\right) - \angle \left(j\frac{\omega}{20} + 1\right)$$

total phase change = -180°

$\angle G(j\omega)$
(degree)



Problem 2.

[10 points]

Suppose the system $H(s) = \frac{1}{s+1}$ is driven by the input signal $x(t) = \cos(5t) + 10 \sin(10t + \pi/3)$. What will be the output signal $y(t)$ at steady state?

$$\cos 5t \rightarrow \boxed{\frac{1}{s+1}} \rightarrow \text{steady state output} = \left| \frac{1}{j5+1} \right| \cos(5t + \angle(\frac{1}{j5+1}))$$

$$= \frac{1}{\sqrt{26}} \cos(5t - 1.373)$$

$$10 \sin(10t + \frac{\pi}{3}) \rightarrow \boxed{\frac{1}{s+1}} \rightarrow \text{steady state output} =$$

$$= 10 \left| \frac{1}{j10+1} \right| \sin(10t + \frac{\pi}{3} + \angle(\frac{1}{j10+1}))$$

$$= \frac{10}{\sqrt{101}} \sin(10t + \frac{\pi}{3} - 1.471)$$

$$\Rightarrow x(t) = \cos 5t + 10 \sin(10t + \frac{\pi}{3}) \rightarrow \boxed{\frac{1}{s+1}} \rightarrow \text{steady state output}$$

$$= \frac{1}{\sqrt{26}} \cos(5t - 1.373) + \frac{10}{\sqrt{101}} \sin(10t - 0.4239)$$

Problem 3.

[5 + 5 = 10 points]

1. What is the natural frequency and damping ratio of $G(s) = \frac{s+20}{2s^2+6s+20}$?

2. Let $G_1(s) = \frac{k}{s^2+2s+10}$ and $G_2(s) = \frac{s+5}{(s+0.02s+0.5)(s^2+2s+50)}$. Determine the value of k so that the steady state outputs of $G_1(s)$ and $G_2(s)$ in response to a unit step input are the same?

$$1. G(s) = \frac{\frac{1}{2}(s+20)}{s^2+3s+10} \Rightarrow \omega_n = 10 \Rightarrow \omega_n = \sqrt{10} = 3.162 \text{ and}$$

$$2\zeta\omega_n = 3 \quad \zeta = \frac{3}{2\sqrt{10}} = 0.4743$$

2. steady state output of $G_1(s)$ in response to unit step input

$$\text{is } 1. G_1(0) = \frac{k}{10}$$

$$\text{steady state output of } G_2(s) \dots \text{ is } 1. G_2(0) = \frac{5}{0.5 \cdot 50} = \frac{1}{5}$$

$$\therefore \text{ we need } \frac{k}{10} = \frac{1}{5} \Rightarrow k = 2 \text{ Ans}$$

Problem 4.

[10 pt]

A second order transfer function $H(s)$ has the following transient response specifications: rise time $t_r \leq 0.1$ sec, peak overshoot $M_p \leq 0.16$, and settling time $t_s \leq 0.511$ sec. Sketch (accurately) the region in the complex plane where the poles of $H(s)$ may lie to meet these specifications.

second order : apply formula

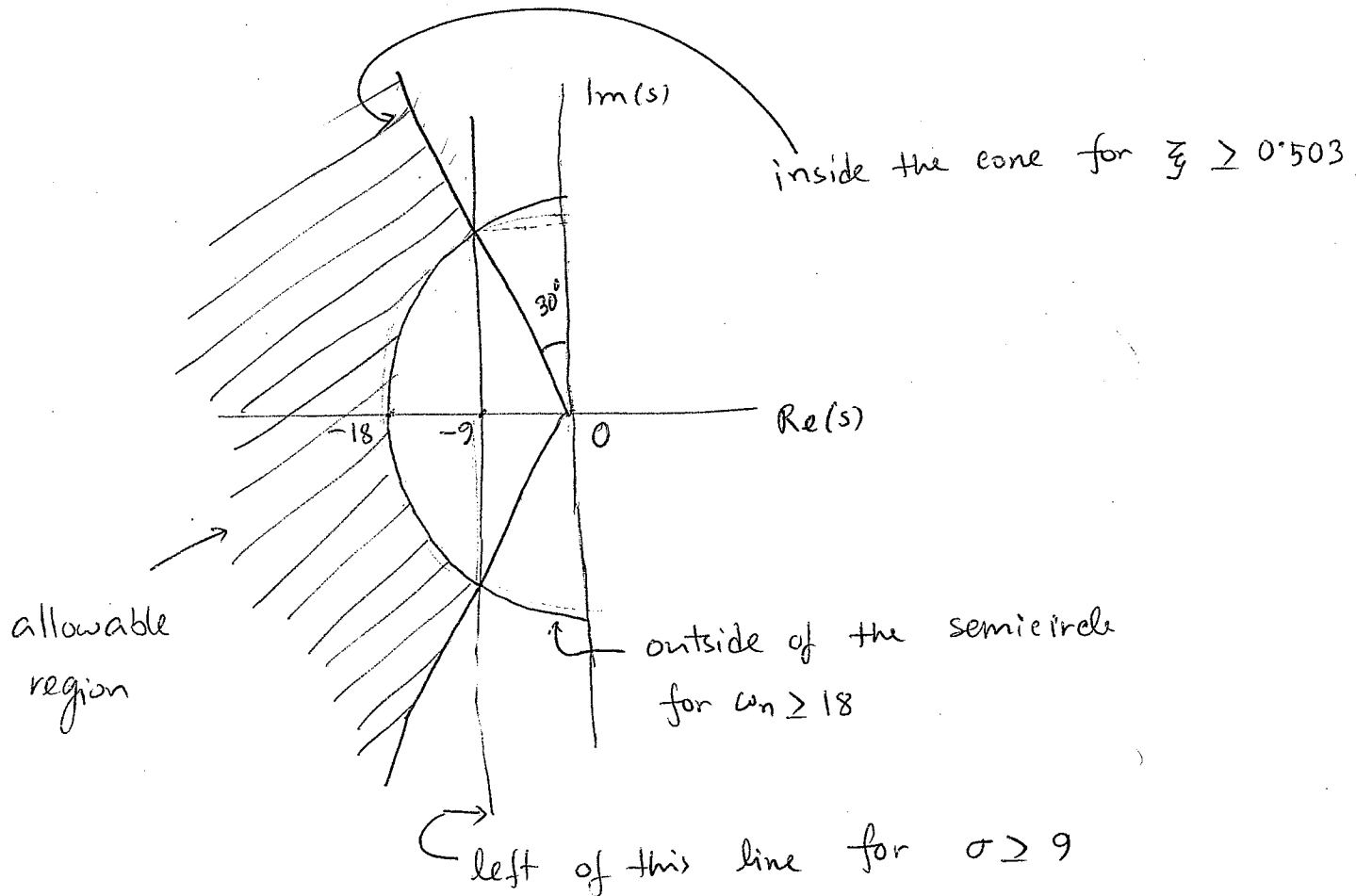
$$\text{need } t_r = \frac{1.8}{\omega_n} \leq 0.1 \Rightarrow \omega_n \geq 18$$

$$t_s \leq \frac{4.6}{\sigma} \leq 0.511 \Rightarrow \sigma \geq 9$$

$$M_p = e^{-\frac{\pi \zeta}{\sqrt{1-\zeta^2}}} \leq 0.16 \Rightarrow \zeta \geq \sqrt{\frac{(\ln 0.16)^2}{\pi^2 + (\ln 0.16)^2}} = 0.503$$

$$\zeta = \sin \theta \Rightarrow \theta \geq 30.2^\circ$$

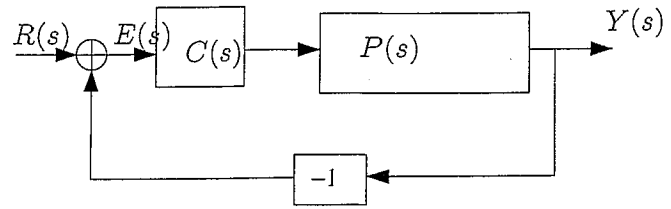
(approximately 30°)



Problem 6.

A proportional controller is to be designed (for the feedback loop shown in the figure) to meet the following transient performance specifications: (i) rise time $t_r \leq 0.2$ sec, (ii) settling time $t_s \leq 10$ sec, and (iii) peak overshoot $M_p \leq 0.1$ in response to a unit step command in the reference $r(t)$. The plant is

$$P(s) = \frac{1}{s^2 + 0.2s + 7}$$



1. Determine the expressions for the damping ratio and natural frequency of the closed loop transfer function from $R(s)$ to $Y(s)$ as a function of the proportional gain k_p .
2. Among the three parameters: rise time, settling time, and peak overshoot of the closed loop, which can be changed by varying k_p ?
3. Determine the minimum value of the proportional gain k_p so that the closed loop rise time specification is met.
4. For this value of k_p , what are the values of the peak overshoot and the settling time of the closed loop?

$$H = \frac{Y}{R} = \frac{PC}{1+PC} = \frac{\frac{1}{s^2+0.2s+7} \cdot k_p}{1 + \frac{1}{s^2+0.2s+7} \cdot k_p}$$

$\left. \begin{array}{l} \text{proportional} \\ \text{controller:} \\ C(s) = k_p \end{array} \right\}$

$$= \frac{k_p}{s^2 + 0.2s + (7+k_p)}$$

1 natural frequency $\omega_n = \sqrt{7+k_p}$
 damping ratio $\zeta = \frac{0.2}{2\omega_n} = \frac{1}{10\omega_n} = \frac{1}{10\sqrt{7+k_p}}$ } Ans.

2 rise time $t_r = \frac{1.8}{\omega_n} = \frac{1.8}{\sqrt{7+k_p}}$ (H can be made to have stable complex poles by choosing $k_p > -6.99$, so these formulas are applicable)

$t_s = \frac{4.6}{\sigma} = \frac{4.6}{0.4}$, $M_p = e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}}$; and ζ is a function of k_p .

So t_r and M_p can be changed by varying k_p
 but t_s cannot be.

3. Need $t_r = \frac{1.8}{\sqrt{7+k_p}} < 0.2 \Rightarrow \sqrt{7+k_p} \geq 9$
 $\Rightarrow k_p \geq \underline{\underline{74}}$ minimum value

4. For $k_p = 74$, $\zeta = \frac{1}{10.9} = \frac{1}{20}$

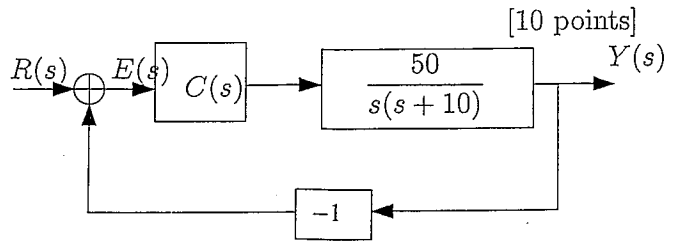
$$\omega_n = 0.2$$

$$\Rightarrow M_p = e^{-\frac{\pi \zeta}{\sqrt{1-\zeta^2}}} = 0.9657$$

and $t_s = \frac{4.6}{\sigma} = \frac{4.6}{.1} = \underline{\underline{46}}$ ms

Problem 7.

Consider the closed loop system shown in the figure, where $C(s) = \frac{s+z}{s+2}$. Plot the (i) real axis part and (ii) the asymptotes of the root locus of the closed loop with respect to the parameter z .



$$H = \frac{Y}{R} = \frac{pe}{1+pe} = \frac{\frac{s+z}{s+2} \cdot \frac{50}{s(s+10)}}{1 + \frac{s+z}{s+2} \cdot \frac{50}{s(s+10)}} = \frac{(s+z)50}{(s+2)(s^2+10s) + (s+z)50}$$

$$\Rightarrow H = \frac{50(s+z)}{s^3 + 12s^2 + 20s + 50s + z \cdot 50} = \frac{50(s+z)}{1 + z \cdot \frac{50}{s^3 + 12s^2 + 70s}}$$

\therefore characteristic eqⁿ is

$$1 + z \cdot \frac{50}{s(s^2 + 12s + 70)} = 0$$

$$k = z,$$

$$L(s) = \frac{50}{s(s^2 + 12s + 70)}$$

poles of $L = 0, -6 \pm 5.83j$

relative degree = 3

real axis part of root locus

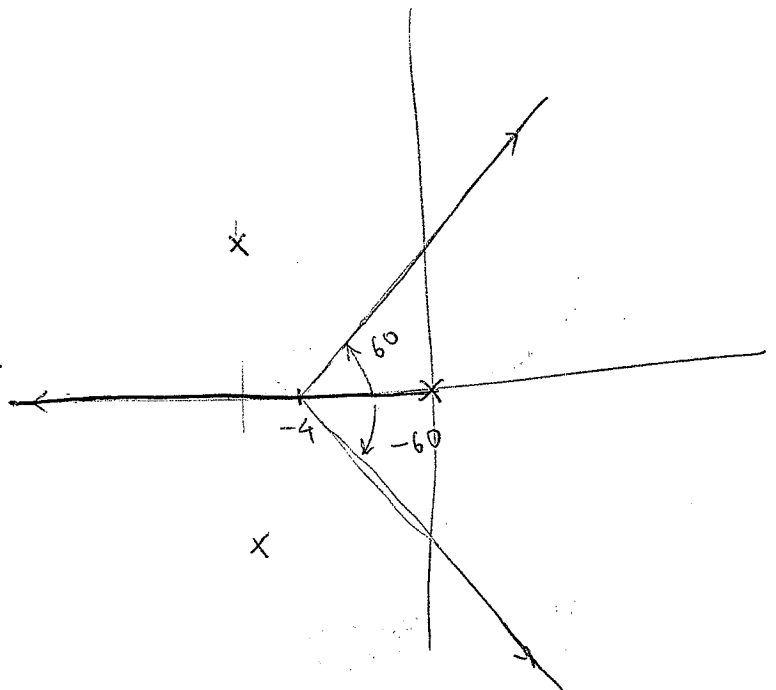
the entire -negative real axis
(odd # of poles)

asymptotes:

$$\alpha = \frac{-12}{3} = -4$$

$$\phi_k = \frac{180}{3}, \frac{180}{3} + \frac{360}{3}, \frac{180}{3} + \frac{360}{3} \cdot 2$$

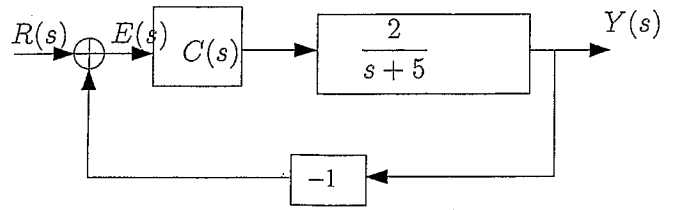
$$= 60, 180, 300$$



Problem 8.

Consider the closed loop control system shown in the figure (similar to the cruise control system in the mini project), where $C(s) = 1 + \frac{k_i}{s}$ is a PI controller. Sketch the root locus of the closed loop poles with respect to the integral gain k_i .

[10 points]



$$H = \frac{Y}{R} = \frac{PC}{1+PC} = \frac{\frac{s+k_i}{s} \cdot \frac{2}{s+5}}{1 + \frac{s+k_i}{s} \cdot \frac{2}{s+5}} = \frac{2(s+k_i)}{s^2 + 5s + 2s + 2k_i}$$

$$\Rightarrow H = \frac{2 \frac{s+k_i}{s(s+7)}}{1 + k_i \frac{2}{s(s+7)}}$$

characteristic eqⁿ: $1 + k_i \frac{2}{s(s+7)} = 0$

$$L(s) = \frac{2}{s(s+7)}$$

zeros: none
poles: 0 and -7
relative degree = 2

$$\Rightarrow \alpha = \frac{-7-0}{2} = -3.5$$

$$\phi_l = \frac{180^\circ}{2} + \frac{360^\circ}{2}(l-1), \quad l=1,2$$

$$\Rightarrow \phi_{1,2} = 90^\circ, 270^\circ$$

